

**TIRE BUILDING DRUM HAVING EXPANDABLE CENTER SECTION  
AND INDEPENDENTLY EXPANDABLE BEAD LOCK ASSEMBLIES  
IN THE END SECTIONS**

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**CROSS-REFERENCE TO RELATED APPLICATIONS**

This application is a continuation-in-part of commonly-owned, copending U.S. Patent Application entitled **EXPANDABLE TIRE BUILDING DRUM WITH ALTERNATING**  
10 **FIXED AND EXPANDABLE SEGMENTS, AND CONTOURS FOR SIDEWALL INSERTS**, Attorney's Docket No. DN2001168USA, and filed on even date herewith.

This application relates to U.S. Patent Application entitled **METHOD FOR MANUFACTURING TIRES ON A FLEXIBLE MANUFACTURING SYSTEM**,  
Attorney's Docket No. DN2001166USA and filed on even date herewith.

15 **TECHNICAL FIELD OF THE INVENTION**

The present invention relates to tire building drums for laying up tire carcasses, more particularly to drums which are expandable between a collapsed position and an expanded position. The invention also relates to methods and apparatus for setting beads on green tire carcasses.

20 **BACKGROUND OF THE INVENTION**

It is known that in making vehicle tires, for example for automobiles, that manufacture of a so-called carcass is first achieved by successively assembling several different components. In other words, the different carcass types included in a production range can be distinguished from one another depending on the presence thereon of the various  
25 accessory components and/or the typology of the accessory components themselves.

By way of example, when carcasses for tubeless tires are to be produced, that is tires that in use do not require the presence of an inner tube, the main components can be considered to include a so-called inner liner that is a layer of elastomeric air-impervious material, a carcass ply, a pair of annular metal elements, commonly referred to as bead cores,  
30 around which the opposite ends of the carcass ply are folded, as well as a pair of sidewalls made of elastomeric material, extending over the carcass ply at laterally opposite positions. The accessory components may in turn comprise of one or more additional carcass plies, one

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or more reinforcing bands for overlying the carcass ply or plies at the areas turned up around the bead cores (chafer strips), and others.

It is well known that the components of most pneumatic tire constructions must be assembled in a way which promotes good tire uniformity in order to provide proper tire performance. For example, a tread which "snakes" as it goes around the tire circumference will cause wobbling as the tire is operated. For example, a carcass ply which is lopsided (longer cords on one side of the tire than the other side) can cause a variety of tire nonuniformity problems including static imbalance and radial force variations. For example, a tire which is not meridionally symmetric (e.g., tread not centered between beads) can cause a variety of tire nonuniformity problems including couple imbalance, lateral force variations, and conicity. Therefore, in order to meet typical tire performance requirements, the tire industry generally expends considerable effort in producing tires with good uniformity. Tire uniformity is generally considered to mean tire dimensions and mass distributions which are uniform and symmetric radially, laterally, circumferentially, and meridionally, thereby producing acceptable results for measurements of tire uniformity including static and dynamic balance, and also including radial force variation, lateral force variation, and tangential force variation as measured on tire uniformity machines which run the tire under load on a road wheel.

Although certain degrees of tire nonuniformity can be corrected in post-assembly manufacturing (e.g., by grinding), and/or in use (e.g., applying balance weights to the rim of a tire/wheel assembly), it is preferable (and generally more efficient) to build-in tire uniformity as much as possible.

Typical tire building machines comprise a tire building drum around which the tire components are wrapped in successive layers including, for example, an innerliner, one or more carcass plies, optional sidewall stiffeners and bead area inserts (e.g., apex), sidewalls and bead wire rings (beads). After this layering, the carcass ply ends are wrapped around the beads, the tires are blown up into a toroidal shape, and the tread/belt package is applied.

Commonly-owned U.S. Patent No. 5,591,288 (hereinafter referred to as "Becker") discloses mechanical tire building drums for building extended mobility pneumatic tires, and more specifically to a tire building drum having contours or depressions in its surface to facilitate building certain tire designs. Attention is also directed to corresponding published European Patent Application No. 0 634 266 A2.

As noted by Becker, tire performance can be affected by adding components to the tire or by adjusting the location of tire components in the tire during the tire building process.

During the tire building process, it is important that components fit together well with a minimum of wrinkling of the tire components or trapping of air between the components. If  
5 air is trapped between the uncured tire components, the tire may be defective and may have to be scrapped. During the tire building process, if it appears the air has been trapped between tire components, the tire builder must stitch the interfaces between the uncured elastomeric components to work any bubbles or trapped air from between the components. This stitching involves rolling a roller wheel along the components, forcing the air to an edge of a  
10 component where it can escape. The stitching process is time consuming and requires the skill of the tire builder.

As further noted by Becker, this problem is further magnified in tire designs where components are rather thick compared to other components. For example, when a component having a relatively square cross-section, such as a tire bead, is positioned adjacent a more  
15 planar component, such as a ply, the air may be trapped where the different-shaped components interface. In tire designs where different-shaped components are necessarily placed next to each other, the problem of trapped air is even more difficult.

As further noted by Becker, in one particular extended mobility tire design, inserts are positioned in the sidewall between the carcass plies to enable the tire to support the weight of  
20 the vehicle even if the tire should lose inflation pressure. These inserts are typically thicker than the plies which lie adjacent to them and it is important that this tire be built without trapping air between the plies and inserts. In accordance with the present invention, an inventive tire building method and drum have been designed which have features to accommodate the special production needs of such tires. These special features will be  
25 described hereinafter and contribute to the building of a quality tire without trapping air.

Becker therefore provides a method of building a tire comprising the steps of forming a liner into a cylinder, positioning first inserts to indent the liner cylindrical surface circumferentially at axially spaced insert locations along the axis of the cylinder, laying a first  
30 ply of reinforcing material around the cylindrical surface of the liner and first insert, positioning second inserts over the first ply at the spaced insert locations, laying a second ply of reinforcing material over the first ply and the second inserts, positioning circular beads at each end of the cylinder, expanding the first ply and the second ply to increase the diameter

of the cylinder between the circular beads to provide shoulders at each end of the cylinder, turning edges of the first ply around the second ply over each of the beads, and positioning a belt and tread assembly around the second ply to form a precured tire.

Becker further provides a method of assembling tire components on a tire building drum having a cylindrical surface comprising the steps of laying a liner on the surface of the drum, positioning first inserts below the cylindrical surface and around a drum at insert locations spaced from each end of the drum, laying a first ply of reinforcing material around the drum over the cylindrical surface of the liner and first insert, positioning second inserts over the first ply at the insert locations spaced from each end of the drum, laying a second ply of reinforcing material over the first ply and the second inserts, positioning circular beads at each end of the drum, expanding the drum to increase the diameter of the cylindrical surface and provide shoulders at each end of the drum, turning edges of the first ply and the second ply over each of the beads, positioning a belt and tread assembly around the second ply, and contracting the drum for removal of the assembled tire components from the drum.

Becker further provides a tire building drum which has a cylindrical surface, circular grooves in the surface at insert locations spaced from each end of the drum for positioning of first inserts below the surface, means for applying a first ply over the cylindrical surface, means for applying second inserts over the first ply and the first inserts, means for applying a second ply over the first ply and second insert, means for expanding the drum providing shoulders at each end of the drum for applying bead rings, means for turning up ends of the first ply around the beads, means for applying a belt and tread assembly around the second ply and means for contracting the drum to remove the assembled tire from the drum.

Commonly-owned U.S. Patent No. 4,855,008 discloses an expandable tire building drum, especially a first stage solid pocket drum for building a carcass of a radial tire, having a segmental drum (10) with a plurality of axially-extending, circumferentially spaced segments (36) with flexible connections (56) to shoulder pistons (32) at opposite ends of each segment (36). Wedge shaped bars (62) are positioned between the segments (36) and are connected to center pistons (64) for urging tapered side faces (80) of the bars into engagement with sloping side faces (78) of the segments (36). The shoulder pistons (32) and center pistons (64) move radially outward to expand the drum. During the first stage operation, the tire reinforcing plies, beads and other components are assembled on the first stage drum and then the carcass is moved to another location where it is shaped and the belt and tread applied. In the first

stage assembly of the tire carcass it is important that the tire components be applied to contracted and expanded drum surfaces which are concentric and of uniform diameter along the length of the drum. Expandable drums of different constructions have been used heretofore; however it has been difficult to maintain a concentric drum surface and a uniform diameter along the length of the drum in both the expanded and contracted condition of the drum. For example, the drum surface may be concentric and uniform in the contracted condition but is distorted during expansion to a larger diameter. As a result, the components added to the carcass on the expanded drum are not precisely assembled which may adversely affect the uniformity of the tire.

U. S. Patent No. 5,264,068 discloses an expandable drum including adjustable stops for setting drum circumference. Tapering structures, each having axial slidability, are provided, and in response to a slide move of the tapering structure, drum segments are each radially expanded or retracted. As noted therein, the tapering structure (12) is of an inner recessed frustum and is mounted over the drum shaft (10) longitudinally or axially slidable with the aid of a key (16), and housed in the drum (14). The drum (14) is circumferentially divided into a plurality of drum segments (17), each being like a sector, and each segment (17) is interiorly supported by a drum segment supporter (18).

Commonly-owned U.S. Patent No. 4,976,804 discloses an expandable, segmental tire building drum (1) having a plurality of circumferentially spaced drum segments (28) radially movable by a set of links (36) pivotally connected to a pair of axially movable hub assemblies (34) slidably mounted on a drum shaft (12). Each of the segments (28) has a cylindrical center portion (30) and end portions (32) with recesses providing pockets (68) for the tire bead portions. The links (36) are positioned between the end portions (32) providing space for large bead portions in the pockets (68) and at the same time the segments (28) are retractable to a small diameter to facilitate placing of a tire band (64) over the drum (10).

Commonly-owned U.S. Patent No. 4,929,298 discloses a tire building drum including an expandable segmental cylinder assembly and a vacuum Chamber. The drum (10) has a plurality of axially-extending, circumferentially spaced segments (18). The ends of the drum are sealed to provide a vacuum chamber (76) inside the drum which is in communication with vacuum holes (78) in a cover sleeve (48) to hold tire components on the drum surface (58) during assembly of the tire components.

## BRIEF SUMMARY OF THE INVENTION

According to the invention, a tire building drum has a center section and two end sections. Each end section is provided with an expandable bead lock assembly. The center section is preferably expandable. The expandable bead lock assembly comprises a carrier ring and a plurality of elongate links extending between the carrier ring and a plurality of radially-expandable segments. When the carrier ring moves inward (towards the center section), the radially-expandable segments move radially outward, urging a plurality of axially extending, circumferentially spaced-apart finger segments outward from a collapsed position to an expanded position, and at least one position therebetween.

In an embodiment of the invention, the bead lock assembly comprises a cylinder and two pistons disposed within the cylinder. The pistons are free to move axially within the cylinder, in response to pneumatic pressure. The first piston is constrained from moving axially inward by rods. The second piston is connected by rods to the carrier ring. Pressurized air supplied through air lines and passageways in the cylinder control the movement of the pistons so that the beach lock assembly can be partially-expanded, fully-expanded, and retracted.

The bead lock assembly of the present invention works well in combination with a tire building drum having an expandable center section. As described herein, a tire building drum has alternating fixed and expanding segments in a center section of the drum. The expanding segments are axially-extending and circumferentially spaced-apart from one another, and their end portions are contoured (have recesses, or grooves) to accommodate tire components such as sidewall inserts. Two different mechanisms for expanding the center section are described. A first mechanism includes two wedge elements which are axially moveable away from one another to expand the center section. Ramp elements associated with the expanding segments may thus be moved radially outward. Biasing elements provide a restoring force for collapsing the center section. A second mechanism includes two guide rings which are axially moveable towards one another for expanding the center section, and away from one another to collapse the center section. Overlapping links are provided between the guide rings and a base member supporting the expanding segments.

According to the invention, there is disclosed a process of building a tire on a tire building drum having an expandable center section and two expandable end sections. The process includes the following steps of first applying an innerliner on a flat application

surface of the tire building drum while the center section and the end sections are in their collapsed conditions. Then the center section and the end sections are expanded to an intermediate expanded condition to form a pair of spaced recesses on the center section of the drum. Next, a pillar insert is applied into each recess of the center section whereby the application surface across the building drum is substantially flat. Continuing, a first ply is applied onto the substantially flat application surface, followed by applying post inserts atop the first ply and substantially above the pillar insert, followed by applying a second ply. Then a pair of beads are moved into place above fingers of a bead lock assembly in each of the expandable end sections. Then, each bead lock assembly and the center section are expanded to their fully-expanded positions so that the fingers grip the inextensible beads. Next, the innerliner, first ply and second ply are turned up about the beads. Continuing, the bead lock assemblies and the center section are collapsed to their collapsed, unexpanded position. Finally, the tire is removed from the drum and the process starts again.

Other objects, features and advantages of the invention will become apparent in light of the following description thereof.

#### **BRIEF DESCRIPTION OF THE DRAWINGS**

Reference will be made in detail to preferred embodiments of the invention, examples of which are illustrated in the accompanying drawing figures. The figures are intended to be illustrative, not limiting. Although the invention is generally described in the context of these preferred embodiments, it should be understood that it is not intended to limit the spirit and scope of the invention to these particular embodiments.

Certain elements in selected ones of the drawings may be illustrated not-to-scale, for illustrative clarity. The cross-sectional views, if any, presented herein may be in the form of "slices", or "near-sighted" cross-sectional views, omitting certain background from lines which would otherwise be visible in a true cross-sectional view, for illustrative clarity.

Elements of the figures are typically numbered as follows. The most significant digit (hundreds) of the reference number corresponds to the figure number. Elements of Figure 1 are typically numbered in the range of 100-199. Elements of Figure 2 are typically numbered in the range of 200-299. Similar elements throughout the drawings may be referred to by similar reference numerals. For example, the element 199 in a figure may be similar, and possibly identical to the element 299 in another figure. Elements of the figures can be numbered such that similar (including identical) elements may be referred to with similar numbers in a single

drawing. For example, each of a plurality of elements collectively referred to as 199 may be referred to individually as 199a, 199b, 199c, etc. Or, related but modified elements may have the same number but are distinguished by primes. For example, 109, 109', and 109" are three different elements which are similar or related in some way, but have significant modifications.

- 5 Such relationships, if any, between similar elements in the same or different figures will become apparent throughout the specification, including, if applicable, in the claims and abstract. Sometimes, similar elements are referred to with the suffixes -L and -R (e.g., 133L, 133R), which generally indicate left and right, as viewed in the drawing.

10 The structure, operation, and advantages of the present preferred embodiment of the invention will become further apparent upon consideration of the following description taken in conjunction with the accompanying drawings, wherein:

**Figure 1A** is a schematic cross-sectional view of a tire building drum, with a tire carcass being laid up thereupon, according to the prior art;

15 **Figure 1B** is a schematic cross-sectional view of a tire building drum, with a tire carcass being laid up thereupon, according to the prior art;

**Figure 2A** is a perspective view of a tire building drum, according to the present invention;

**Figure 2B** is a perspective view of a center section of the tire building drum of **Figure 2A**, in a collapsed position (condition), according to the invention;

20 **Figure 2C** is a cross-sectional view of the center section shown in **Figure 2B**, according to the invention;

**Figure 2D** is a perspective view of a center section of the tire building drum of **Figure 2A**, in an expanded position (condition), according to the invention;

25 **Figure 2E** is a cross-sectional view of the center section shown in **Figure 2D**, according to the invention;

**Figure 2F** is a perspective view of a typical expanding segment of the center section of the tire building drum of **Figure 2A**, according to the invention;

**Figure 3A** is a perspective view of the center section of a tire building drum, according to an embodiment of the invention;

30 **Figure 3B** is a cross-sectional view of the center section of **Figure 3A**, in a fully-collapsed condition;



**Figure 3C** is a cross-sectional view of the center section of **Figure 3A**, in a semi-expanded condition;

**Figure 3D** is a cross-sectional view of the center section of **Figure 3A**, in a fully-expanded (or semi-collapsed) condition;

5        **Figure 4A** is a perspective view of the center section of a tire building drum, according to an alternate embodiment of the invention, showing the center section in a fully-collapsed condition;

10       **Figure 4B** is a perspective view of the center section of a tire building drum, according to an alternate embodiment of the invention, showing the center section in a fully-expanded condition;

**Figure 4C** is a schematic illustration of how the linkage mechanism of the alternate embodiment of **Figure 4A** and **Figure 4B** works, according to the invention;

**Figure 4D** is a plan view of an alternate embodiment of a component of the linkage mechanism, according to the invention;

15       **Figure 5** is a partial cross-sectional view of a tire carcass laid up on a tire to submit building drum, according to the invention;

**Figure 6A** is a schematic plan view of a prior art bead holder which can be used in conjunction with practicing the method of the present invention, in a closed position;

**Figure 6B** is a schematic plan view of the bead holder of **Figure 6A**, in the open position;

20       **Figure 7** is a detailed cross-sectional view of a tire building drum with expandable end sections, according to the invention;

**Figure 7A** is a, cross-sectional view of an end section of the tire building drum of **Figure 7**, in a fully-collapsed position (condition);

25       **Figure 7B** is an end cross-sectional view of the end section shown in **Figure 7A**, taken on a line 7B-7B through **Figure 7A**;

**Figure 8A** is a perspective view of the outer end of the end section of the tire building drum of **Figure 7**;

**Figure 8B** is a, cross-sectional view, similar to that of **Figure 7A**, showing details of a stop mechanism for the pistons within the end section;

30       **Figure 9A** is a, cross-sectional view, similar to that of **Figure 7A** or **Figure 8B**, illustrating the end section of the tire building drum of **Figure 7**, in a semi-expanded (or semi-collapsed) condition;

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**Figure 9B** is a perspective view of the outer end of the end section of **Figure 9A**, in the semi-expanded (or semi-collapsed) position (condition);

**Figure 10A** is a, cross-sectional view, similar to that of **Figure 7A** or **Figure 8B**, illustrating the end section of the tire building drum of **Figure 7**, in a fully-expanded

5 condition; and

**Figure 10B** is a perspective view of the inner end of the end section (722) of **Figure 9A**, in the fully-expanded position (condition).

## DEFINITIONS

The following terms may be used throughout the descriptions presented herein and should generally be given the following meaning unless contradicted or elaborated upon by other descriptions set forth herein.

"Apex" (also "Bead Apex") refers to an elastomeric filler located radially above the bead core and between the plies and the turnup plies.

"Axial" and "axially" refers to directions that are on or are parallel to the tire's axis of rotation.

"Axial" refers to a direction parallel to the axis of rotation of the tire.

"Bead" refers to that part of the tire comprising an annular substantially inextensible tensile member, typically comprising a cable of steel filaments encased in rubber material.

"Belt structure" or "reinforcement belts" or "belt package" refers to at least two annular layers or plies of parallel cords, woven or unwoven, underlying the tread, unanchored to the bead, and having both left and right cord angles in the range from 18 to 30 degrees relative to the equatorial plane of the tire.

"Breakers" or "tire breakers" refers to a belt or belt structure or reinforcement belts.

"Carcass" refers to the tire structure apart from the belt structure, tread, undertread over the plies and the sidewalls, but including the beads, plies, and, in the case of EMT or runflat tires, the wedge inserts sidewall reinforcements.

"Casing" refers to the carcass, belt structure, beads, sidewalls and all other components of the tire excepting the tread and undertread.

"Centerplane" refers to a plane intersecting a line which is normal to the plane at a point which is midway between two other points on the line. The line may be an axis of a cylindrical member, such as a tire building drum. A finished tire has a centerplane which is the "equatorial plane" of the tire.

"Chafer" refers to reinforcing material (rubber alone, or fabric and rubber) around the bead in the rim flange area to prevent chafing of the tire by the rim parts.

"Chipper" refers to a narrow band of fabric or steel cords located in the bead area whose function is to reinforce the bead area and stabilize the radially inwardmost part of the sidewall.

5 "Circumferential" refers to circular lines or directions extending along the perimeter of the surface of the annular tread perpendicular to the axial direction, and can also refer to the direction of sets of adjacent circular curves whose radii define the axial curvature of the tread, as viewed in cross section.

"Cord" refers to one of the reinforcement strands, including fibers or metal or fabric,  
10 with which the plies and belts are reinforced.

"Crown" or "tire crown" refers to the tread, tread shoulders and the immediately adjacent portions of the sidewalls.

"EMT tire" refers to Extended Mobility Technology and EMT tire refers to a tire which is a "runflat", which refers to a tire that is designed provide at least limited operational service  
15 under conditions when the tire has little to no inflation pressure.

"Equatorial plane" refers to a the plane perpendicular to the tire's axis of rotation and passing through the center of its tread, or midway between the tire's beads.

"Gauge" refers generally to a measurement, and often to a thickness dimension.

"Inner liner" refers to the layer or layers of elastomer or other material that form the  
20 inside surface of a tubeless tire and that contain the inflating gas or fluid within the tire. Halobutyl, which is highly impermeable to air.

"Insert" refers to the crescent-shaped or wedge-shaped reinforcement typically used to reinforce the sidewalls of runflat-type tires; it also refers to the elastomeric non-crescent-shaped insert that underlies the tread; it is also called a "wedge insert."

25 "Lateral" refers to a direction parallel to the axial direction.

"Meridional profile" refers to a tire profile cut along a plane that includes the tire axis.

"Ply" refers to a cord-reinforced carcass reinforcing member (layer) of rubber-coated radially deployed or otherwise parallel cords.

"Pneumatic tire" refers to a laminated mechanical device of generally toroidal shape  
30 (usually an open-torus) having two beads, two sidewalls and a tread and made of rubber, chemicals, fabric and steel or other materials.

"Shoulder" refers to the upper portion of sidewall just below the tread edge.

"Sidewall" refers to that portion of a tire between the tread and the bead.

"Tire axis" refers to the tire's axis of rotation when the tire is mounted to a wheel rim and is rotating.

"Tread cap" refers to the tread and the underlying material into which the tread pattern is molded.

"Turn-up end" refers to a portion of a carcass ply that turns upward (i.e., radially outward) from the beads about which the ply is wrapped.

## DETAILED DESCRIPTION OF THE INVENTION

Generally speaking, a conventional process for making a radial-ply automobile tire includes an intermediate step of disposing two annular inextensible beads, each comprising a cable of steel filaments encased in green rubber, over the other components of a green ("green" meaning as yet uncured and still tacky) tire carcass on a tire building drum. An annular cross-sectionally triangular rubber filler called an "apex" may be added. Portions of the ply components that extend beyond the beads are then turned up around the beads, forming "turn-ups". Then, the green carcass is typically removed from the tire building drum and mounted on a "second stage machine" where it is inflated (reshaped) to a toroidal shape, and its radially-outer surface is pressed against a tread and belt package. In subsequent steps, the green carcass is stitched (rolled with a roller) to remove air pockets and adhere internal surfaces together. The resulting assembly is inserted into a mold (vulcanizing press) to cure under heat (typically 350 degrees Fahrenheit) and pressure to become a finished tire.

**Figure 1A** corresponds generally to FIG. 9 of Becker, and illustrates (schematically, and in a greatly simplified manner) an exemplary tire building drum 102 of the prior art. The drum 102 is generally cylindrical, having two ends 102a and 102b, an axis of rotation 104 extending between the two ends, and a cylindrical outer surface 106. A centerplane (**CP**) is indicated on the drawing, and is generally a plane which bisects a carcass being laid up on the tire building drum.

In a typical (again, greatly simplified, for illustrative clarity) tire buildup, an inner liner 108 is applied on the surface of the drum 102, and two tire sidewall insert components ("inserts") 110a and 110b (collectively referred to as "110") are disposed at longitudinally (axially) spaced apart positions on the inner liner 108, as shown. Next, a first ply 112 is disposed over the inner liner 108 and inserts 110. This results in a green tire carcass having a nominally cylindrical shape. However, as is evident from the illustration of **Figure 1A**, the addition of the sidewall inserts 110 between the inner liner 108 and the ply 112 causes there

to be two "bumps" (protrusions), which are regions of increased outside diameter ("OD"), in the outer surface of the carcass. As can be seen, these bumps protrude significantly upwardly from the outer surface of the tire building drum and create significant protrusions 18 in these areas. Subsequent tire components such as a second carcass ply are difficult to force into such a nonplanar contour. At the locations of the protrusions, air can be trapped within the tire, leading to the aforementioned problems.

Next, two beads 114a and 114b (collectively "114") are added to the tire carcass. Each bead 114 is a substantially inextensible circular hoop, having an inside diameter ("ID") which is substantially equal to or preferably only slightly greater than the OD of the ply 112 (in areas other than where there are bumps). The beads 114 are shown as being slightly axially outboard of the inserts 110, and are shown as having a round (versus hexagonal) cross-section for sake of illustrative clarity. A second ply (not shown) may be added to the carcass, and the outer end portions of the carcass may be turned up. Finally the carcass may be transferred to another (second stage) machine for adding a tread package, etc.

**Figure 1B** corresponds generally to FIGs. 2-7 of Becker, and illustrates an alternate embodiment of an exemplary tire building drum 122 of the prior art. The drum 122 is generally cylindrical, having two ends 122a and 122b, an axis of rotation 124, and a generally cylindrical outer surface 126. The drum 122 differs from the drum 102 of **Figure 1A** primarily by virtue of having annular recesses (pockets, grooves) 136a and 136b (collectively referred to as "136") in its outer surface at longitudinal (axial) positions corresponding to the positions of and related to the dimensions of inserts 130a and 130b (collectively referred to as "130") and extending about the circumference of the drum 122. In this example, the inner liner 128 is applied to the surface 126 of the drum 122. Then the inserts 130 are applied, and fit (nestle) down into the recesses 136. Then a ply 132 is applied. This results in a green tire carcass having a substantially cylindrical shape. In contrast to the tire carcass formed in Figure 1A, the addition of the inserts 130 between the inner liner 128 and the ply 132 does not cause there to be two "bumps" in the outer surface of the carcass. Since there are substantially no bumps, and the outer surface of the tire carcass being laid up is substantially cylindrical, having a substantially uniform OD, it is (among other things) possible to mount two beads 134a and 134b (collectively referred to as "134") onto the carcass by sliding them both on from one end (e.g., 122a) of the drum 122.

**Figures 2A through 2D** illustrate, generally, the tire building drum 202 of the present invention. The drum 202 is generally cylindrical, having two ends 202a and 202b, an axis of rotation 204 extending between the two ends, and a cylindrical outer surface 206. The drum 202 has an overall axial length " $L$ " between the two ends. A spindle (or drum support shaft) extends along the axis 204 and has an end 208a extending from the end 202a of the drum 202, and an end 208b extending from the end 202b of the drum 202.

The drum 202 has a center section 220 which is generally cylindrical, and centered about the axis 204. The center section 220 has a width (more properly, axial length) of  $L_C$ . The drum 202 has a first end section 222 which is coaxial with the center section 220, and which is disposed axially at one end of the center section 220. The drum 202 has a second end section 224 which is coaxial with the center section 220, and which is disposed axially at an opposite end of the center section 220. The two end sections 222 and 224 are, for purposes of the present invention, substantially identical to (i.e., mirror images of) one another, each having an axial length of  $(L - L_C)/2$ . The end sections 222 and 224 are axially-outward of the center section 220. The drum, more significantly the center section 220 of the drum, has a centerplane (compare **CP, Figure 1A**), which is a plane intersecting the axis 204 midway between the ends of the center section (typically also midway between the ends 202a, 202b) of the overall drum. The axis 204 is, by definition, normal to the centerplane.

The center section 220 is circumferentially segmented, having a plurality of elongate fixed segments 226 alternating with a like plurality of elongate expanding segments 228. As best viewed in any of **Figures 2B-2D**, there are suitably 24 (twenty four) fixed segments 226 alternating with 24 (twenty four) expanding segments 228. The expanding segments 228 are axially-extending and circumferentially spaced from one another, and end portions of each is contoured to have annular recesses (pockets, grooves) 236a and 236b (collectively referred to as "236"; compare 136) in its outer surface at longitudinal (axial) positions corresponding to the positions of and related to the dimensions of sidewall inserts (not shown, compare 130) which will be applied during the carcass layup process, described hereinabove. The pockets 236 can best be viewed in **Figure 2F**, wherein can also be viewed two turnup bladder (not shown) anchor points 238a and 238b in the outer surface of the expanding segment. In **Figures 2F and 5A**, it is seen that end portions of the expanding segments 238, 538 are contoured to have pockets 236, 536 for receiving components (e.g., sidewall inserts) of a tire carcass being laid up on the drum.

The fixed segments 226 are elongate, generally rectangular in cross-section and have a length substantially equal to  $L_C$ . The fixed segments 226 typically have a fixed width or have a width proportional to the number of total segments. The expanding segments 228 are also elongate, generally rectangular in cross section, a length substantially equal to  $L_C$ , and typically have a fixed width or have a width proportional to the number of total segments. The expanding segments 228 are also elongate, generally rectangular in cross section, and have a length substantially equal to  $L_C$ .

It is within the scope of the invention that there are any suitable number of fixed and expanding segments, for example, rather than twenty four of each, anywhere from eighteen to thirty of each. It is also within the scope of the invention that the number of fixed segments is not exactly equal to the number of expanding segments. It is also within the scope of the invention that the expanding segments do not all have the exact same width. The same applies to the fixed segments. Selected ones of the fixed and/or expanding segments can be "special purpose" segments, such as for communicating vacuum to an inner liner being laid up on the drum.

The center section 220 is expandable, between a collapsed (or retracted, or contracted) condition, shown in **Figures 2B and 2C** and an expanded (or extended) condition (or "fully" expanded position), shown in **Figures 2D and 2E**. Mechanisms for effecting expansion and collapse of the center section 220 are described hereinbelow, and accommodate partially expanding the center section to one (or more) "semi-expanded" positions. Generally, each of said expanding segments 228 is expandable from a first drum radius in the a collapsed condition of the drum to a second, greater drum radius in a semi-expanded expanded condition of the drum and finally drum to a third drum radius, greater than the second radius in a fully expanded condition of the drum.

#### **"Dual Cone" Mechanism For Expanding/Collapsing The Center Section**

**Figures 3A-3D** illustrate the major components of an expandable center section 320 (compare 220) of a tire building drum, according to an embodiment of the invention. In the view of **Figure 3A**, one of a plurality (e.g., 24) of expanding segments 328 (compare 228) is shown, and a corresponding one of a plurality (e.g., 24) of fixed segments 326 (compare 226) is shown. In **Figures 3B-3D**, the expanding segment 328 is shown, but not the fixed segment 326, for illustrative clarity. A spindle 308 is illustrated highly schematically in **Figures 3B-3D**, and is omitted from **Figure 3A**, for illustrative clarity. A base member 346 for the fixed

segment 326 is shown in **Figure 3A** only, for illustrative clarity. A base (ramp) element 348 for the expanding segment 328 is best viewed in **Figures 3B-3D**.

Two guide elements (flanges) 340a and 340b (collectively referred to as "340") are disposed at axially spaced apart positions on a 308 (compare 208) which extends along the axis 304. The flanges 340 are suitably in the form of generally planar discs which are centered on the axis 304, and are parallel with one another. Each flange 340 has an inner surface which faces, and is parallel with the inner surface of the other flange 340. The flanges 340 are essentially fixed to the spindle 308, which means that they will rotate with the spindle, and that they are at a fixed axial distance apart from one another. The flanges 340 are preferably centered about the centerplane. The flanges 340 are a distance apart which, as illustrated, is less than the length  $L_c$  of the segments 326, 328.

The inner surfaces of the flanges 340a and 340b are provided with a plurality of radially-extending grooves 342a and 342b, respectively. A given groove 342a on the guide plate 340a corresponds to, and is at the same circumferential position on the spindle as, a given groove 342b on the guide plate 340b. These two given grooves 342a, 342b form a given pair of grooves and, for example, there are 24 (twenty four) pairs of grooves, spaced at even intervals about the inner surfaces of the flanges 340. Each of these given pairs of grooves functions as a track for guiding an expanding segment support member (ramp element) 348 associated with an expanding segment 328, radially inward and outward, as discussed hereinbelow.

Each expanding segment 328 has a ramp element 348 associated therewith. (For 24 expanding segments 328, there are 24 ramp elements 348.) The ramp element 348 is essentially a flat planar element having four edges (sides) - a top edge for supporting the expanding segment 328, a bottom "ramped" edge which functions as a ramp surface for being acted upon by two movable wedge elements 358 (described in greater detail hereinbelow), a first side edge which rides in the groove 342a of a given groove pair, and a second side edge which rides in the groove 342b of the given groove pair. Preferably, the ramp element 348 is separate from the expanding segment 328, but it is within the scope of the invention that it is integrally formed therewith. In the case that the ramp element 348 is not formed integrally with the expanding segment 328, the expanding segment 328 may be attached in any suitable manner to the ramp element 348.



The inner surfaces of the flanges 340a and 340b are also provided with a plurality of radially-extending grooves 343a and 343b, respectively. Each of the radially-extending grooves 343a and 343b are interspersed between the radially-extending grooves 342a and 342b. The radially-extending grooves 343a and 343b are shorter than the radially-extending grooves 342a and 342b. A given groove 343a on the guide plate 340a corresponds to, and is at the same circumferential position on the spindle 308 as, a given groove 343b on the guide plate 340b. These two given grooves 343a, 343b form a given pair of grooves and, for example, there are 24 (twenty four) pairs of grooves, spaced at even intervals about the inner surfaces of the flanges 340. Each of these given pairs of grooves 343a, 343b function as a track for receiving and securing a fixed segment support member 346 associated with a fixed segment 326, as discussed hereinbelow. The base member 346 is essentially a rectangular block, extending between grooves of the flanges and having four edges (sides) - a top edge for supporting the fixed segment 326, a first side edge which fits in a groove 343a, a second side edge which fits in a groove 343b, and a generally flat bottom edge. In the case of 24 (twenty four) fixed segments 326, there are 24 (twenty four) base members 346 extending between 24 pairs of grooves 343a, 343b. (The side edges of the base members are received in the grooves.) This accounts for the total overall number of grooves in each flange (and the total overall number of groove pairs in the flanges) being 48 (forty eight) - 24 pairs of grooves for guiding the expanding segments 328 as they move radially in and out, and 24 pairs of grooves for locating the fixed segments 326 between the expanding segments 328 even though radial movement is not contemplated or required (to the contrary, the fixed segments are supposed to remain at selected radial positions). Preferably, the base member 346 is separate from the fixed segment 326, but it is within the scope of the invention that it is integrally formed therewith. In the case that the base member 346 is not formed integrally with the fixed segment 326, the fixed segment 326 may be attached in any suitable manner to the base member 346.

Each fixed segment 326 has a base member 326 associated therewith.. (For 24 fixed segments 326, there are 24 base members 346.) The base member 346 is essentially a rectangular block, extending between grooves of the flanges and having four edges (sides) - a top edge for supporting the fixed segment 326, a first side edge which fits in a groove 342a, a second side edge which fits in a groove 342b, and a generally flat bottom edge. In the case of 24 (twenty four) fixed segments 326, there are 24 (twenty four) base members 346 extending

between 24 pairs of grooves. (The side edges of the base members are received in the grooves.) This accounts for the total overall number of grooves in each flange (and the total overall number of groove pairs in the flanges) being 48 (forty eight) - 24 pairs of grooves for guiding the expanding segments 328 as they move radially in and out, and 24 pairs of grooves for locating the fixed segments 326 between the expanding segments 328 even though radial movement is not contemplated or required (to the contrary, the fixed segments are supposed to remain at selected radial positions). Preferably, the base member 346 is separate from the fixed segment 326, but it is within the scope of the invention that it is integrally formed therewith. In the case that the base member 346 is not formed integrally with the fixed segment 326, the fixed segment 326 may be attached in any suitable manner to the base member 346.

In **Figure 3A**, it can be seen that the fixed segment 326 has an axial length which is substantially the same as the axial length of the expanding segment 328, and that the axial length  $L_C$  of both is greater than the spacing between the two flanges 340, and that they are "centered" with regard to the flanges 340 (and the centerplane).

Two biasing members 338a and 338b (collectively referred to as "338") are provided. One of the biasing members, 338b) is shown in phantom in **Figure 3A**. The other of the biasing members, 338a, is shown in phantom in **Figures 3B-3D**, for illustrative clarity. The biasing members 338 are disposed at axially spaced apart positions about the spindle 308, and are suitably in the form of rubber bands extending through corresponding holes **342a** and **342b** in each of the ramp elements 348. These rubber bands 338 exert a "collapsing" radial force on the ramp elements 348 in the direction of the axis 304. As shown in **Figure 3A**, the base members 346 for the fixed segments 326 may also be provided with holes 344a and 344b, through which the rubber bands 338 extend.

Two tapered (wedge) elements 358a and 358b (collectively referred to as "358") are disposed at axially spaced apart positions on the spindle 308 (on either side of the centerplane). The wedge elements 358 are suitably in the form of generally planar discs (rings, since they are discs with a hole in the middle) which are centered on the axis 304, and are parallel with one another. The outer faces of the wedge elements 358 are tapered. Therefore, the wedge elements 358 are frustoconical, and may be referred to as "cones", or "cone-shaped elements", or "conical elements". The wedge elements 358 are not fixed to the spindle 308. Rather, although they may be keyed (or splined) to the spindle so that they will

rotate with the spindle, they are free to move axially (traverse) along the spindle, towards and apart from one another, from a minimum distance (essentially touching one another), to a maximum distance from one another, remaining parallel with each other irrespective of the axial distance from one another.

5           In **Figure 3B**, the center section 320 is shown in its collapsed (or "fully-collapsed") position. In this position, the wedge elements 358 are close together (e.g., essentially zero distance apart from one another, with their bases touching, or nearly touching), and the ramp element 348 and, consequently, the expanding segment 328 is at its minimal radial distance from the axis 304. In other words, the diameter of the center section 320 is at a minimum in  
10 this collapsed (retracted) position. In this collapsed position, the outer surface of the center section 320 has substantially the same diameter as that of the outer surfaces 306 (compare 206) of adjacent end sections 322 and 324 (compare 222, 224). In this collapsed position, a tire component, such as the inner liner (e.g., 504, see below) of a tire carcass, may be applied.

          In **Figure 3C**, the center section 320 is shown in its semi-expanded position. In this  
15 position, the wedge elements 358 are spread apart from one another (but not as far apart as they are capable of spreading), and the ramp element 348 and, consequently, the expanding segment 328 is at a greater radial distance from the axis 304. In other words, the diameter of the center section 320 is now larger, or expanded. In this semi-expanded position, the outer surface of the center section 320 has a slightly greater diameter than that of the outer surfaces  
20 306 (compare 206) of adjacent end sections 322 and 324 (compare 222, 224). In this semi-expanded position, a tire component, such as the ply (e.g., 508, see below) of a tire carcass, may be applied.

          In **Figure 3D**, the center section 320 is shown in its fully-expanded position. In this position, the wedge elements 358 are spread (have moved) farther apart from one another  
25 (essentially as far apart as they are capable of spreading, their bases far apart from one another), and the ramp element 348 and, consequently, the expanding segment 328 is at an even greater radial distance from the axis 304. In other words, the diameter of the center section 320 is now even larger, or more expanded. In this fully-expanded position, the outer surface of the center section 320 has a much greater diameter than that of the outer surfaces  
30 306 (compare 206) of adjacent end sections 322 and 324 (compare 222, 224). In this fully-expanded position, the beads are caused to be firmly set to the carcass, the turnup ends of which may then be turned up, in a final step of carcass construction. Then, the center section

320 of the drum can be partially collapsed (e.g., returned to a semi-expanded position), and the carcass can be removed for further processing, such as the application of a tread package in a second stage tire building machine.

The two wedge elements 358 are in the form of cones (more accurately, frustroconical), disposed coaxially (having the same axis) with their bases opposing (facing) one another, and their apexes (albeit truncated) remote from one another. It is preferred that the two wedge elements 358 remain at all times, throughout their range of axial movement, equidistant from the centerplane of the center section 320 of the drum. The bottom edge (inner surface) of the ramp element 348 is V-shaped, with two intersecting ramp surfaces, one for each of the wedge elements 358. In this manner, forces exerted by the wedge elements 358 are evenly distributed along the length of the ramp element 348 and, consequently, the expanding segment 328. The angle along the outer edges (faces) of the wedge elements 358, and the corresponding angle along the inner edges (surfaces) of the ramp elements 348 is suitably between 20 degrees and 45 degrees, such as approximately 30 degrees, more particularly such as 33 degrees, with respect to the axis, or more parallel to the axis than perpendicular thereto. This angle, of course, remains constant irrespective of the axial positions of the wedge elements 358. As the wedge elements 358 move farther apart from one another, the expanding segments 328 are urged radially outward from the axis 304.

The expanding segment 328 has a length  $L_C$ . The fixed segment 326 has a length substantially equal to  $L_C$ . The flanges 340 are spaced apart a distance less than the length  $L_C$ . In the illustrations of **Figures 3A-3D**, a total of 48 (forty eight) grooves 342 are shown in each flange 340. As discussed hereinabove, 24 of these grooves on each flange form a given pair of grooves for guiding the ramp elements 348 as they are forced radially outward and return radially inward. As best viewed in **Figure 3A**, the base member 346 extends between intermediate pairs of grooves 342 in the flanges 340. Also, the base members 346 must pass over (by, through) the wedge elements 358. Therefore, the wedge elements 358 have 24 notches 356 at evenly spaced circumferential positions about the outer surface of their respective bases for receiving a bottom edge of the base member 346 as it passes by. This serves to 'lock' the wedge elements 358 in fixed circumferential positional relationship with respect to the flanges 340, while allowing the wedge elements 358 to move axially back and forth in the space between the flange elements 340.

It is therefore seen that expansion of the center section 320 of a tire building drum can be accomplished using a traversing dual cone mechanism which exerts radial forces on the expanding segments 328 which are symmetrical about the centerplane of the drum (i.e., of the center section 320). With only one tapering structure, such as in USP 5,264,068, such

5 symmetry cannot be accomplished. Applying expanding forces, with symmetry about the centerplane, can be critical to achieving uniformity in the layup of a tire carcass.

Although not shown, any suitable mechanism can be used for moving the tapered wedge elements axially 358 outward to effect expansion of the center section 320, and axially inward (towards one another) for permitting retraction of the center section 320.

10 Suitable dimensions for the center section 320 are:

- diameter collapsed = 400 mm
- diameter semi-expanded = 420 mm
- diameter fully-expanded 476 mm (expansion of 76 mm)
- minimum center section width ( $L_c$ ) of 250 mm

15 When the center section 320 is collapsed, the surface of the drum is substantially continuous, smooth, uninterrupted (flat), and this is advantageous for innerliner application. It is within the scope of the invention that means for providing a vacuum, through selected ones of the segments (either fixed or expanding), to the surface of the drum, to hold the innerliner securely thereon, be provided, in any suitable manner. When the center section is semi-expanded, the

20 surface is also substantially flat, such as would be advantageous for ply application.

#### **"Overlapping Linkage" Mechanism For Expanding/Collapsing The Center Section**

**Figures 4A-4C** illustrate an alternate embodiment of a mechanism for expanding and collapsing the center section of a tire building drum. Whereas the embodiment of **Figures 3A-3D** used a dual cone and ramp mechanism for expansion, and rubber bands for collapsing

25 the center section, in this embodiment the linkage is capable of both expanding and contracting the expanding segments of the center section.

**Figures 4A-4C** illustrate the major components of an expandable center section 420 (compare 320) of a tire building drum, according to an alternate embodiment of the invention. In the illustration of **Figure 4C**, one of a plurality (e.g., 24) of expanding segments 428

30 (compare 328) is shown. In the views of **Figures 4A and 4B**, the expanding segment is omitted, for illustrative clarity. It will be understood that the general alternating arrangement of fixed and expanding segments is substantially the same in this embodiment as in the

previously-described embodiment. In describing this embodiment, the fully-collapsed position of the center section 420 is shown in **Figure 4A**, and the fully-expanded position of the center section 420 is shown in **Figure 4B**. It will be understood that in this, as in the previous embodiment, the drum may be expanded (or collapsed) to any position (diameter) between fully-collapsed and fully-expanded. A spindle (compare 308) extends along the axis 404 of the drum, but it omitted, for illustrative clarity. Although not shown, the center section is provided with fixed segments (e.g., 326), in the same (or similar) manner as was the previously-described embodiment.

Two flanges 440a and 440b (collectively referred to as "440"; compare 340) are disposed at axially spaced apart positions on the spindle. The flanges 440 are substantially similar to the flanges 340 of the previous embodiment, and are suitably in the form of generally planar discs which are centered on the axis (304), and are parallel with one another. Each guide element 440 has an inner surface which faces, and is parallel with the inner surface of the other guide element 440. The flanges 440 are essentially fixed to the spindle (308), which means that they will rotate with the spindle (308), and that they are at a fixed axial distance apart from one another.

The inner surfaces of the flanges 440a and 440b are provided with a plurality of radially-extending grooves 442a and 442b and interspersed grooves 443a and 443b, respectively. Again, this is comparable to the grooves 342a and 342b and 343a and 343b of the previously-described embodiment. A given groove 442a on the guide plate 440a corresponds to, and is at the same circumferential position on the spindle as, a given groove 442b on the guide plate 440b. These two given grooves 442a, 442b form a pair of grooves and, for example, there are 24 pairs of grooves, spaced at even intervals about the inner surfaces of the flanges. Each pair of grooves functions as a track for guiding an expanding segment support, or base (support) element 448 (compare 348) as it moves radially inward or outward from the axis, as discussed hereinbelow.

Each expanding segment 428 has a support element 448 associated therewith. (For 24 expanding segments, there are 24 base members.) The support element 448 is essentially a flat planar element having four edges (sides) - a top edge for supporting the expanding segment 328, a first side edge which rides in the groove 442a of a given groove pair, and a second side edge which rides in the groove 442b of the given groove pair. The support element 448 also has a bottom edge, but the shape of that edge is of no particular importance

(as contrasted with the bottom edge ramp surface of the ramp element 348). Preferably, the support element 448 is separate from the expanding segment 428, but it is within the scope of the invention that it is integrally formed therewith. In the case that the support element 448 is not formed integrally with the expanding segment 428, the expanding segment 428 may be attached in any suitable manner to the support element 448.

Two guide rings (hubs) 458a and 458b (collectively referred to as "458") are disposed at axially spaced apart positions on the spindle (on either side of the centerplane). The guide rings 458 are suitably in the form of generally planar discs (rings, since they are discs with a hole in the middle) which are centered on the axis 404, and are parallel with one another. The guide rings 458 are not fixed to the spindle. Rather, although they may be keyed (or splined) to the spindle so that they will rotate with the spindle, they are free to move axially along the spindle, towards and apart from one another, from a minimum distance (essentially touching one another), to a maximum distance from one another, remaining parallel with each other irrespective of the axial distance from one another.

An overlapping linkage mechanism 460 is provided between the guide rings 458 and the support element 448. The linkage mechanism comprises:

a first elongate link 462 having an end pivotally attached to a one (458a; left, as viewed) of the guide rings 458, and an opposite end pivotally attached adjacent (near) a one (right, as viewed) end of the support element 448; and

a second elongate link 464 having an end pivotally attached to the other (458b; right, as viewed) of the guide rings 458, and an opposite end pivotally attached adjacent (near) an opposite (left, as viewed) end of the support element 448.

The links 462 and 464 overlap each other (cross over one another), but are not pivotally attached to each other, as would be the case with a "scissors" type linkage, nor are they parallel to each other, as would be the case with a two-link "toggle" type linkage.

In **Figure 4A** (compare **Figure 3B**) the center section 420 is shown in its collapsed (or "fully-collapsed") position. In this position, the guide rings 458 are spread far apart from one another (essentially as far apart as they are capable of spreading), and the support element 448 and, consequently, the expanding segment 428 is at its minimal radial distance from the axis 404. In other words, the diameter of the center section 420 is at a minimum in this collapsed position. In this collapsed position, the outer surface of the center section 420 has

substantially the same diameter as that of the outer surfaces (306) of adjacent end sections (322, 324). In this collapsed position, the inner liner of a tire carcass may be applied.

In **Figure 4B** (compare **Figure 3D**), the center section 420 is shown in its fully-expanded position. In this position, the guide rings 458 are close together (e.g., essentially zero distance apart from one another), and the support element 448 and, consequently, the expanding segment 428 is at its greatest greater radial distance from the axis 404. In other words, the center section 420 is now fully-expanded. In this fully-expanded position, the outer surface of the center section 420 has a much greater diameter than that of the outer surfaces (306) of adjacent end sections (e.g., 222, 224). Concurrently with the drum in the fully-expanded position, separately actuated bead locks (not shown) cause the beads to be firmly set. Next, the ends of the carcass can then be turned up, in a final step of carcass construction. Then, the center section 420 of the drum can be partially collapsed (e.g., returned to a semi-expanded position), the bead locks collapsed and the carcass can be removed for further processing, such as the application of a tread package in a second stage tire building machine.

In the collapsed condition (**Figure 4A**), the links 462 and 464 are both nearly parallel to the axis 404. For example, at an angle of 19.6 degrees with respect thereto. In the expanded condition (**Figure 4B**) the links 462 and 464 are at an angle approximately halfway between parallel to and perpendicular to the axis 303, such as at an angle of 46.2 degrees with respect thereto. This provides for a relatively compact mechanism with a good operating range.

Although not shown, the center section can be expanded to any diameter between collapsed and fully-expanded, as determined by the spacing of the guide rings 458 from one another. For example, in a semi-expanded position, the ply of a tire carcass may be applied. It is preferred that the two guide rings 458 remain equidistant from the centerplane of the center section 420 of the drum while they are moving in their range of positions. In this manner, forces are evenly (symmetrically) distributed along the length ( $L_c$ ) of the support element 448 and the expanding segment 428.

In this example, with the overlapping linkage, the relationship between guide ring spacing and center section diameter is inverse - the closer the guide rings are to one another, the greater the diameter of the center section. In the previous example (wedge/ramp), the relationship between guide rings spacing and center section diameter is direct - the closer the



guide rings are to one another, the lesser the diameter of the center section. In either case however, the diameter of the center section 320 and 420 is proportional (directly or inversely, respectively) to the spacing between the wedge elements 358 or guide rings 458, respectively.

The overlapping linkage mechanism of **Figures 4A-4C** is superior to a toggle linkage, for example as shown in the aforementioned USP 4,929,298 with regard to being able to apply forces to the expanding segment in a manner which is symmetrical about the centerplane, throughout the range of expansion for the drum. A toggle linkage, wherein two links move in unison parallel to one another, is inherently not symmetrical about the centerplane. This symmetry, as in the previous (wedge) embodiment, can be of profound significance in achieving uniformity in the layup of the tire carcass.

The overlapping linkage embodiment of **Figures 4A-4C** is similar to the wedge/ramp embodiment of **Figures 3A-3D**, in the following regards:

- both are for expanding and collapsing a center section (220, 320, 420) of a tire building drum;
- both act upon expanding segments (228, 328, 428) of the center section;
- both do not act upon the fixed segments (226, 326, 426) of the center section;
- both employ flanges (340, 440) which have grooves (342, 442) for guiding a ramp element (348) or support element (448) which supports the expanding segment (328, 428);
- both have elements (358, 458) which move axially to effect the expansion/collapse of the center section;
- both exert expanding forces on the expanding segments in a manner which is symmetrical about the centerplane.

The symmetry of forces exerted (urged) upon the expanding segments, about the centerplane, is non-trivial. As mentioned above, a carcass ply which is lopsided (longer cords on one side of the tire than the other side) can cause a variety of tire nonuniformity problems including static imbalance and radial force variations. The present invention addresses one potential source of such nonuniformities - namely, imprecise (e.g., non-cylindrical) expansion of the drum.

In both embodiments, when the center section (320, 420) is collapsed, the surface of the drum is substantially continuous, smooth, uninterrupted (flat), and this is advantageous for innerliner application. It is within the scope of the invention that means for providing a vacuum, through selected ones of the segments (either fixed or expanding), to the surface of

the drum, to hold the innerliner securely thereon, be provided, in any suitable manner. When the center section is semi-expanded, the surface is also substantially flat, such as would be advantageous for ply application. Both embodiments can use a roller screw system for center section expansion. The mechanism for moving the wedges 358 or guide rings 458 depends largely on other factors present in the overall drum construction, and can be adapted on a case-by-case basis.

The overlapping linkage embodiment of **Figures 4A-4C** is different from the wedge/ramp embodiment of **Figures 3A-3D**, in the following regards:

- in the wedge/ramp embodiment, rubber bands 338 are used to collapse the center section 320;
- in the overlapping linkage, the links 462, 464 themselves effect collapse of the center section;
- in the wedge/ramp embodiment, the center section 320 expands when the wedges 358 move axially apart, and retracts when the wedges 358 move together.
- in the overlapping linkage, the center section 420 expands when the guide rings 458 move closer together, and retracts when the guide rings 458 move farther apart.

The overlapping linkage design tends to provide more expansion range in a narrower width ( $L_C$ ), allowing the minimum drum width to shrink, for example from 250mm (for the wedge embodiment) to 200 mm (for the linkage embodiment) .

Some exemplary dimensions for the center section 420 of the linkage embodiment are presented in the following table.

Tire Size (in.)	14	15	16	17	18	19	20
Rim Dia. (in.)	14	15	16	17.2	18.2	19.2	20.2
Expanded (mm)	391	416	441	472	497	523	548
Intermediate (mm)	338	364	390	420	444	468	493
Collapsed (mm)	308	334	350	380	404	428	453
expansion (mm)	83	82	91	92	93	95	95

**Figure 4D** illustrates an alternate embodiment of a support element 448' which is provided with two holes 442a and 442b (compare 342a and 342b) for receiving biasing members

comparable to the biasing members 338 shown in **Figures 3A-3D**. The biasing members, suitably in the form of rubber bands, would exert a "collapsing" radial force on the support element 448'.

### **Extended Mobility Tires**

**Figure 5** is a partial cross-sectional view of an exemplary tire carcass as it is laid up on a tire building drum, according to the invention. An end of an expanding segment 528 is shown. First, a center sleeve 502 is installed on the surface of the drum and extends over the expanding segment 528. . An upper turnout bladder 503 and a lower turnout bladder 505 extends beyond the drum. The tire carcass comprises the following major components, in the following order:

- an innerliner 504;
- a first sidewall insert (pillar) 506;
- a first ply (ply 1) 508;
- a second sidewall insert (post) 510;
- a second ply (ply 2) 512;
- a bead 514;
- an apex 516;
- a chafer 518; and
- a sidewall 520.

Other components, such as chipper, gum toeguard and fabric toeguard may be added to the carcass, as may be desired, but form no special part of the present invention.

### **Mounting Beads On A Tire Carcass**

**Figures 1A and 1B** illustrated beads 114 and 134 in place on a tire carcass being laid up on tire building drums 102 and 122, respectively. As mentioned above, each bead 114 and 134 is a substantially inextensible, circular hoop, having an inside diameter ("ID") which is substantially equal to or preferably only slightly greater than the OD of the ply 112 or 132, respectively. The beads 114 and 134 are shown as being installed slightly axially outboard of the inserts 110 and 130, respectively.

**Figures 2A through 2D** illustrated a tire building drum 202 which is generally cylindrical, having two ends 202a and 202b, an axis of rotation 204 extending between the two ends, and a cylindrical outer surface 206. As mentioned above, the drum 202 has a center section 220 which is generally cylindrical, and centered about the axis 204. The drum

202 has a first end section 222 which is coaxial with the center section 220, and which is disposed axially at one end of the center section 220. The drum 202 has a second end section 224 which is coaxial with the center section 220, and which is disposed axially at an opposite end of the center section 220. The two end sections 222 and 224 are substantially identical to  
 5 (i.e., mirror images of) one another. The end sections 222 and 224 are axially-outward of the center section 220

Generally, the two beads for a tire may be installed, one from each end of the drum, over the carcass being laid up, onto a respective end section thereof. As described in the figures that follow, the two end sections of a tire build drum are expandable, for "setting" the  
 10 beads. Therefore, each end section includes a "bead lock assembly" which expands for setting the bead disposed on that section. This is discussed in greater detail hereinbelow.

It will also be recalled that the center section of the drum is expandable, having (for example) a plurality of elongate fixed segments 226 alternating with a like plurality of elongate expanding segments 228. The beads are typically moved onto the respective end  
 15 sections of the drum by using a bead holder which holds the bead and moves it into position about the carcass being laid up. The position at which the two beads are installed is approximately at the inner edge of the respective end section.

**Figures 6A and 6B** illustrate a bead holder 622 in a closed and open position, respectively. The bead holder 622 comprises a support (base) 602 and a ring 604. The ring  
 20 604 has an inside diameter " $d$ ". The ring 604 comprises three segments - a left segment 604a, a middle segment 604b and a right segment 604c. The three segments 604a, 604b and 604c are typically of equal arcuate extent - namely, approximately 120 degrees each. The middle segment 604b is fixed to the support 602. The left and right segments 604a and 604c are pivotally affixed to the middle segment 604b (as shown), or directly to the support 602.

25 A mechanism 606 is provided for causing the left segment 604a to pivot from its closed position (**Figure 6A**) to its open position (**Figure 6B**). A mechanism 607 is provided for causing the right segment 604a to pivot from its closed position (**Figure 6A**) to its open position (**Figure 6B**). In the open position, the distal ends of the left and right segments 604a and 604c are spaced a distance apart " $e$ " which is greater than the diameter (OD) of a tire  
 30 drum (more particularly, of a carcass being laid up upon the drum), so that it can be removed from the drum simply by raising it (radially, with respect to the drum) off of the drum. This

radial direction for removing the open bead holder 622 from a drum (not shown) having an axis 634 is indicated by the arrow 636.

A plurality of magnets 608 are disposed just inside inner edge of the ring 604. These magnets are for holding a bead 612 (shown only partially, for illustrative clarity) onto the ring 604. The magnets 608 are strong enough to hold the bead 612, but weak enough to let the bead 612 stay in place on a drum, or on a tire carcass being laid up on the drum when the bead holder 622 is removed from the drum.

As described hereinabove with respect to **Figure 5**, a center sleeve 502 is installed on the surface of the drum and extends over the expanding segment 528 of the center section of the drum. Upper and lower turnup bladders 503 and 505 extend beyond an adjacent end section of the drum. The construction and operation of the turnup bladders are described in greater detail hereinbelow.

### **Tire Building Drum with Expandable End Sections**

**Figure 7** illustrates a tire building drum 700 (compare 202). The drum 700 is generally cylindrical, having two ends (compare 202a, 202b), an axis 704 (compare 204), and a generally cylindrical outer surface 706 (compare 206). The drum 700 has an overall axial length (compare  $L$ ) between the two ends. The drum 700 has a center section 720 (compare 220) which is generally cylindrical, and centered about the axis 704. The center section 720 has a width (compare  $L_c$ ). The drum 700 has a first end section 722 (compare 222) which is coaxial with the center section 720, and which is disposed axially at one end of the center section 720. The drum 700 has a second end section 724 (compare 224) which is coaxial with the center section 720, and which is disposed axially at an opposite end of the center section 720. The two end sections 722 and 724 are, for purposes of the present invention, substantially identical to (i.e., mirror images of) one another.

As described hereinabove with respect to **Figure 2**, the center section 720 is suitably circumferentially segmented, having a plurality of elongate fixed segments (not shown; compare 226) alternating with a like plurality of elongate expanding segments 728 (compare 228). The expanding segments 728 are axially-extending and circumferentially spaced from one another, and end portions of each is contoured to have annular recesses (pockets, grooves; compare 236a and 236b) in its outer surface at longitudinal (axial) positions corresponding to the positions of and related to the dimensions of sidewall inserts (e.g., 506, 510) which will be applied during the carcass layup process. The expanding segments 728

also have anchor points (compare 238a and 238b) for anchoring a center sleeve 713a, 713b which extends to the bladder 714a, 714b, respectively. Although the present invention is not limited to any particular dimensions, exemplary dimensions for a tire building drum are set forth hereinabove. Exemplary dimensions for the end sections 722 and 724 may be

5 extrapolated from dimensions of the center section 720, based on **Figure 7**.

As discussed hereinabove, the center section 720 (220) is suitably expandable, between a collapsed (or retracted, or contracted) condition, and an expanded (or extended) condition (or "fully" expanded position), and various mechanisms for effecting expansion and collapse of the center section are described hereinabove, and the mechanisms accommodate

10 partially expanding the center section to one (or more) "semi-expanded" (or semi-collapsed) positions. It has been discussed hereinabove that different tire components may be applied onto the tire carcass being laid up on the drum at different expansion positions (conditions) of the center section.

**Figure 7** illustrates two end sections 722 and 724, one at each end of the center section 720. The end sections 722, 724 are provided with expandable bead lock assemblies 726a, 726b, including mechanisms for expanding the bead lock assemblies for applying selected tire components (e.g., beads) onto the tire carcass being laid up on the drum at different expansion positions (conditions) of the end sections, as described in greater detail hereinbelow. Since the two end sections are essentially mirror-images of one another, it is

20 sufficient to describe only one of the end sections in detail.

**Figure 7** also illustrates turnup bladders disposed on the end sections 722, 724. A bottom turnup bladder 712a is disposed on the outer surface of the end section 722. A bottom turnup bladder 712b is disposed on the outer surface of the end section 724. A top turnup bladder 714a is disposed over the bottom turnup bladder 712a on the outer surface of the end section 722. A top turnup bladder 714b is disposed over the bottom turnup bladder 712b on the outer surface of the end section 724. As is generally well known, the turnup bladders 712a/b and 714a/b are for turning up turnup ends of the green carcass around respective beads 734a and 734b (compare 134a and 134b; also 512).

#### **Expandable Bead Lock Assemblies**

30 Each of the end sections 722 and 724 is provided with a bead lock assembly 726a, 726b (collectively "726"). Since the end sections 722 and 724 are substantially "mirror

images" of one another, it is sufficient to describe the bead lock assembly 726 of only a single end section 722 in detail.

As illustrated in **Figure 7**, the end section 722 is provided with a bead lock assembly 726a comprises the following major components:

- 5       - a first piston "**P1**";
- second piston "**P2**";
- a carrier ring "**CR**";
- a plurality of elongate segments "**S**"; and
- a plurality of elongate links (link arms) "**K**".

10       The pistons **P1** and **P2** are each generally in the form of flat discs, both centered on the axis 704 (hence, "coaxial"), and each having substantially the same outer diameter as the other. The axis 704 is normal to the planes of the pistons **P1** and **P2**. The pistons **P1** and **P2** are disposed in a cylinder block (or simply "cylinder") 730, a cylindrical interior portion 732 ("piston portion") of which has an inner diameter corresponding to the outer diameter of the

15       pistons **P1** and **P2**. The pistons **P1** and **P2** are disposed in this piston portion 732 of the cylinder 730, and are free to move axially inward and outward (with reference to the center section 720) therein.

Appropriate seals at the outer edges of the pistons **P1** and **P2** are provided since, as will be described in greater detail hereinbelow, the pistons **P1** and **P2** are moved axially inward and

20       outward by selective application of air (or hydraulic) pressure to their inner (towards the center section 720) or outer (away from the center section 720) faces.

The first piston **P1** is disposed axially outward (away from the center section 720) of the second piston **P2**. The second piston **P2** is thus disposed axially inward of the first piston **P1**. In **Figure 7**, the two pistons **P1** and **P2** are shown abutting one another, and the bead

25       lock assembly 726 is in its collapsed position. As described in greater detail hereinbelow, the two pistons **P1** and **P2** are axially movable, and when they move, they impart axial movement to the carrier ring **CR**. The plurality of links (link arms) "**K**" extend between the carrier ring **CR** and radially inner ends of the expandable segments **S**. One end of a link **K** is pivotably connected to the carrier ring **CR**, the other end is pivotably connected to a radially

30       inner end of an expandable segment **S**. The expandable segments **S** are constrained from axial movement, and limited to radial movement. When the carrier ring **CR** moves axially inward (towards the center section 720), the expandable segments **S** move radially outwardly.

In a corollary manner, when the carrier ring **CR** moves axially outward (sway from the center section), the expandable segments **S** move radially inwardly. The expandable segments **S** are elongate, and suitably substantially square in cross-section (see, e.g., **Figure 10B**).

5 An end plate 734 is disposed at the outer end of the cylinder 730 - more particularly, at the outer end of the piston portion 732 of the cylinder 730. This end plate 734 defines the outer end of the piston portion 732, closing it off and establishing a limit to outward movement of the pistons **P1** and **P2**. It also seals off the outer end of the piston portion 732. An annular projection 736 extends from the inner surface of the cylinder 730 at a position  
10 spaced axially inward from the end plate 734, and defines an inner end of the piston portion 732. This annular projection 736 establishes a limit to inward movement of the pistons **P1** and **P2**. The pistons **P1** and **P2** are free to move axially, in the piston portion 732 of the cylinder, between the end plate 734 and the annular projection 736. In this manner, an airtight piston portion 732 is defined.

15 Two pneumatic (e.g., air) lines 742 and 744 are shown in **Figure 7**, both of which have an end terminating in the end plate 734, are disposed at the outer end of the cylinder 730. As described hereinbelow, pressure in these lines 742 and 744, in conjunction with a third line 745 (best seen in **Figure 8A**) control movement of the pistons **P1** and **P2**.

As shown in **Figure 7**, pneumatic line 744 directs pressurized air through air  
20 passageway **PW1** behind piston **P1**. Pneumatic line 742 directs pressurized air through air passageway **PW2** between pistons **P1** and **P2**. Although not shown, pneumatic line 742 directs pressurized air through an unseen passageway air **PW3** between piston **P2** and annular projection 736.

As mentioned above, the expandable segments **S** are constrained from axial  
25 movement, and are limited to radial movement. As illustrated in **Figure 7**, the expandable segments **S** move radially in a radial channel which is formed between the inward (towards the center section 720) end 730a of a cylinder 731a and an end plate 723a at the inward end 722a of the end section 722. The expandable segments **S** are suitably in the form of square elongate shafts. Finger segments "**F**" which are circumferential segments are disposed at  
30 radially outer ends of the expandable segments **S**. There are a plurality, such as twelve, expandable segments **S**, and a like plurality, such as twelve, finger segments **F**. The finger



segments **F** are spaced partially, such as at about 30 degrees, about the circumference of the end section 722.

The invention of an expandable bead lock assembly 726 for an end sections 722 and 724 of a tire building drum 700 was described, generally, with respect to **Figure 7**. In the following figures, details of the operation of the bead lock assembly are shown, including with the bead lock assembly in different positions or conditions (e.g., collapsed, partially expanded, fully-expanded).

**Figures 7A, 7B, 8A and 8B** illustrate the end section 722 of the tire building drum 700, in a fully-collapsed condition. This is somewhat analogous to the situation where, in **Figure 3**, the center section (220) was shown in a fully-collapsed condition. **Figures 9A and 9B** illustrate the end section 722 of the tire building drum 700, in a semi-expanded (or semi-collapsed) condition. This is somewhat analogous to the situation where, in **Figure 3C**, the center section (220) was shown in a semi-expanded (or semi-collapsed) condition. **Figures 10A and 10B** illustrate the end section 722 of the tire building drum 700, in a fully-expanded condition. This is somewhat analogous to the situation where, in **Figure 3D**, the center section (220) was shown in a fully-expanded condition.

As described above, the mechanical components of the bead lock assembly 726 include:

- a first piston "**P1**";
- second piston "**P2**";
- a carrier ring "**CR**";
- a plurality of radially expandable segments "**S**"; and
- a plurality of elongate links (link arms) "**K**"; and
- a plurality of finger segments "**F**".

The bead lock assembly 726 further comprises the following mechanical components:

- three rods **R1P1, R2P1, R3P1** associated with piston **P1**;
- stop blocks **B1** associated with the three rods **R1P1, R2P1, R3P1**; and
- three rods **R1P2, R2P2, R3P2** connecting piston **P2** to **CR**.

Three pneumatic lines 742, 743 and 744 are provided, along with associated passageways **PW1, PW2 and PW3** in the cylinder block (730) for providing pressurized air at the following locations:

- to the outer side of piston **P1**, for moving the piston **P1** inward;

- between piston **P1** and piston **P2**, for moving the piston **P2** inward; and
- to the inner side of piston **P2** for retracting the pistons **P1** and **P2** causing the bead lock assembly to retract.

A cycle of usage is now described, starting with the bead lock assembly 726 retracted (with the end section 722 in its collapsed condition). This is best viewed in **Figures 7A, 7B, 8A and 8B**. The pistons **P1** and **P2** are in their outermost position, with the piston **P1** butted up against the endplate 734, and the piston **P2** butted up against the piston **P1**. The expandable segments **S** are in their retracted position, as are the finger segments **F**. The finger segments **F** are at a first radius with respect to the centerline through the drum. The end section 722 is in a condition of minimum diameter.

As best viewed in **Figures 9A and 9B**, in a first expansion step (semi-expanded), pressurized air is provided through the line 744, through the passageway **PW1** to the outer surface of the piston **P1**. This causes the piston **P1** to move axially inward, towards the center section 720. When the piston **P1** moves inward, it pushes the piston **P2** inward. Inward axial movement of the piston **P1** is limited by three rods **R1P1, R2P1, R3P1** extending through the end plate 734 into the piston **P1**, as described below. Three rods **R1P2, R2P2, R3P2** extend axially between piston **P2** and the carrier ring **CR**. Therefore, when the piston **P2** moves inward, the carrier ring **CR** moves inward. The elongate links **K** extend between the carrier ring **CR** and the expandable segments **S**. When the carrier ring **CR** moves inward, the expanding segments **S** move radially outward. The plurality of finger segments **F** are disposed at the outer ends of the elongate expanding segments **S**. When the finger segments **F** move radially outward to a second radius, larger than the first, the diameter of the bead lock assembly 726 in the end section 722 is increased. Thus, when pressurized air is provided in the line 744, the bead lock assembly 726 becomes partially expanded.

The three rods **R1P1, R2P1, R3P1** extend through the end plate 734 into the piston **P1**, preferably at evenly spaced circumferential positions (120 degrees) about the axis 704. In conjunction with a stop block **B1**, these rods limit the inward axial movement of the piston **P1**. This is the intermediate, partially-expanded condition of the bead lock assembly 726. To adjust the intermediate position, different length stop blocks **B1** can be used.

As best viewed in **Figures 10A and 10B**, further expansion of the bead lock assembly is accomplished by providing pressurized air through the line 742 into the passageway **PW2** which is between the two pistons **P1** and **P2**. This causes the piston **P2** to move further

inward, thereby via the rods **R1P2**, **R2P2**, **R3P2**, moving the carrier ring **CR** inward.

Further inward movement of the carrier ring **CR**, causes the links **K** to move the expandable segments **S** and finger segments **F** radially outward to a third radius larger than the second, thereby increasing the diameter of the bead lock assembly 726 to its fully-expanded

condition. In this step, the piston **P1** may normally will retract (move axially outward until stopped by the end plate 734), as illustrated in **Figure 10A**.

Retraction of the bead lock assembly 726 is accomplished by providing pressurized air through the line 743 into the passageway **PW3** (see **Figure 7A**) to the axially inward side of the piston **P2**. At the same time the pressurized air in lines 742 and 744 is stopped. The pressurized air in line 743 causes the piston **P2** to move axially outward, thereby via the rods **R1P2**, **R2P2**, **R3P2** moving the carrier ring **CR** axially outward, thereby via the links **K** moving the expandable segments **S** and finger segments **F** radially inward, thereby decreasing the diameter of the bead lock assembly 726 to its fully-collapsed condition. The piston **P2** moves axially outward until it is stopped by the piston **P1**. If, in the previous step, the piston **P1** had been restrained from retracting, in its intermediate position, it could be selectively maintained in the unretracted position, and the axially outward movement of the piston **P2** would be limited by the piston **P1**, thereby establishing a partially-collapsed condition for the bead lock assembly, after which by allowing the piston **P1** to retract fully, the piston **P2** could move further axially outward to allow the bead lock assembly to achieve its fully- collapsed condition.

It is thus evident that the end section 722 (and, of course, the end section 724) can selectively and controllably be expanded and collapsed. Unlike the center section 720, essentially the entire outer surface of which can be expanded and collapsed, it is only a small segment of the end sections 722,724 that is expanded and collapsed, namely a band defined by the plurality of finger segments **F**. The band defined by the segments **F** extends axially from an inner end of the respective end section 722,724 towards the outer end thereof, and circumferentially entirely around the end section. The finger segments **F**, hence the band, is expandable from a first radius in a collapsed condition of the end sections 722,724 of the drum to a second radius in an intermediate, partially-expanded condition and then to a third radius in a fully expanded condition of the end sections of the drum.

To summarize the expansion/contraction of the bead lock assembly 726, pressurized air supplied through the first passage 744 via the first passageway **PW1** to an outer side of the

first piston **P1** causes the first piston **P1** to move axially inward, pushing the second piston **P2** also axially inward, until constrained by the rods **R1P1,R2P1,R3P1**, so that the bead lock assembly 726 is partially-expanded. Pressurized air supplied through the second passage 742 via the second passageway **PW2** to a location between the inner side of the first piston **P1** and the outer side of the second piston **P2** causes the second piston **P2** to move further axially inward, until stopped by a projection 736, so that the bead lock assembly 726 is fully-expanded. Pressurized air supplied through the third passage 743 via the third passageway **PW3** to a location on the inner side of the second piston **P2** causes the second piston **P2** to move axially outward, so that the bead lock assembly 726 is fully collapsed unless stopped by the first piston **P1** as discussed above.

### Process Flow

There is now described an exemplary sequence of operations for laying up a tire carcass, accounting for expansion of both the center section 720 and the end sections 722 and 724.

(a) First, in the collapsed position (see, e.g., **Figures 3A, 3B, 4A, 7A, 7B, 8A, 8B**), the innerliner 504 is applied over the centersleeves 713a,713b which ensure a flat application surface.

(b) Next, both the center section 720 and the end sections 722 and 724 are expanded to an intermediate condition, so that there is a flat surface across the entire drum. (see, e.g., **Figures 3C, 9A, 9B**)

- Then, in the intermediate condition, the pillar insert 506 is applied into the recess 236 on the expandable segment 228 of the center section 220.
- Then, in the intermediate condition, the first ply 508 is applied.
- Then, in the intermediate condition, the post insert 510 is applied, atop the first ply 508 and substantially above the pillar insert 506.
- Then, in the intermediate condition, the second ply 512 is applied.
- (c) Next, the beads 514, 716a, 716b are moved into place with a bead holding apparatus 622, and held above the fingers **F** of the bead lock assemblies 726.
- (d) Next, the bead lock assemblies 726 are expanded, and the center section 720 is also be expanded, both to the fully-expanded position so that the fingers **F** grip the inextensible beads. The beads clamp down on the ends of the upper turnup bladders 714a,714b and form a seal.

(e) Then the upper turnup bladders 714a,714b are inflated and begin the turnup of the tire components about the beads 514, 716a, 716b.

(f) Continuing, the bottom turnup bladders 712a,712b are inflated to complete turnup of tire components about the beads.

5 (g) The sidewalls can then be applied to the carcass while the drum and beadlock assemblies are in the fully expanded position.

(g) Then, the bead lock assemblies 726 and the center section 720 are collapsed. Note that the bead lock assemblies are forced to collapse due to the positive unlocking of the fingers **F** because of the air moving the piston **P2** away from the center of the drum.

10 (h) Finally, a transfer ring can move over the beads of the tire carcass. A vacuum draws the carcass away from the drum and the green tire carcass is removed from the drum.

Although the invention has been illustrated and described in detail in the drawings and foregoing description, the same is to be considered as illustrative and not restrictive in character - it being understood that only preferred embodiments have been shown and described, and that all  
15 changes and modifications that come within the spirit of the invention are desired to be protected.

Undoubtedly, many other "variations" on the "themes" set forth hereinabove will occur to one having ordinary skill in the art to which the present invention most nearly pertains, and such variations are intended to be within the scope of the invention, as disclosed herein.

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